Sliding Mode Methodology Vs. Computed Torque Methodology Using MATLAB/SIMULINK and Their Integration into Graduate Nonlinear Control Courses

Farzin Piltan SSP.ROBOTIC@yahoo.com

Industrial Electrical and Electronic Engineering SanatkadeheSabze Pasargad. CO (S.S.P. Co), NO:16, PO.Code 71347-66773, Fourth floor Dena Apr, Seven Tir Ave, Shiraz, Iran

Sajad Rahmdel SSP.ROBOTIC@yahoo.com

Industrial Electrical and Electronic Engineering SanatkadeheSabze Pasargad. CO (S.S.P. Co), NO:16 ,PO.Code 71347-66773, Fourth floor Dena Apr, Seven Tir Ave, Shiraz, Iran

Saleh Mehrara SSP.ROBOTIC@yahoo.com

Industrial Electrical and Electronic Engineering SanatkadeheSabze Pasargad. CO (S.S.P. Co), NO:16 ,PO.Code 71347-66773, Fourth floor Dena Apr, Seven Tir Ave, Shiraz, Iran

Reza Bayat SSP.ROBOTIC@yahoo.com

Industrial Electrical and Electronic Engineering SanatkadeheSabze Pasargad. CO (S.S.P. Co), NO:16 ,PO.Code 71347-66773, Fourth floor Dena Apr, Seven Tir Ave, Shiraz, Iran

Abstract

Design a nonlinear controller for second order nonlinear uncertain dynamical systems is one of the most important challenging works. This paper focuses on the design, implementation and analysis of a chattering free sliding mode controller for highly nonlinear dynamic PUMA robot manipulator and compare to computed torque controller, in presence of uncertainties. In order to provide high performance nonlinear methodology, sliding mode controller and computed torque controller are selected. Pure sliding mode controller and computed torque controller can be used to control of partly known nonlinear dynamic parameters of robot manipulator. Conversely, pure sliding mode controller is used in many applications; it has an important drawback namely; chattering phenomenon which it can causes some problems such as saturation and heat the mechanical parts of robot manipulators or drivers. In order to reduce the chattering this research is used the linear saturation function boundary layer method instead of switching function method in pure sliding mode controller. These simulation models are developed as a part of a software laboratory to support and enhance graduate/undergraduate robotics courses, nonlinear control courses and MATLAB/SIMULINK courses at research and development company (SSP Co.) research center, Shiraz, Iran.

Keywords: MATLAB/SIMULINK, PUMA 560 Robot Manipulator, Nonlinear Position Control Method, Sliding Mode Control, Computed Torque Control, Chattering Free, and Nonlinear Control.

1. INTRODUCTION

Computer modeling, simulation and implementation tools have been widely used to support and develop nonlinear control, robotics, and MATLAB/SIMULINK courses. MATLAB with its toolboxes such as SIMULINK [1] is one of the most accepted software packages used by researchers to enhance teaching the transient and steady-state characteristics of control and robotic courses [3_7]. In an effort to modeling and implement robotics, nonlinear control and advanced MATLAB/SIMULINK courses at research and development SSP Co., authors have developed MATLAB/SIMULINK models for learn the basic information in field of nonlinear control and industrial robot manipulator [8, 9].

Controller is a device which can sense information from linear or nonlinear system (e.g., robot manipulator) to improve the systems performance [3]. The main targets in designing control systems are stability, good disturbance rejection, and small tracking error[5]. Several industrial robot manipulators are controlled by linear methodologies (e.g., Proportional-Derivative (PD) controller, Proportional- Integral (PI) controller or Proportional- Integral-Derivative (PID) controller), but when robot manipulator works with various payloads and have uncertainty in dynamic models this technique has limitations. From the control point of view, uncertainty is divided into two main groups: uncertainty in unstructured inputs (e.g., noise, disturbance) and uncertainty in structure dynamics (e.g., payload, parameter variations). In some applications robot manipulators are used in an unknown and unstructured environment, therefore strong mathematical tools used in new control methodologies to design nonlinear robust controller with an acceptable performance (e.g., minimum error, good trajectory, disturbance rejection [10-18].

Sliding mode controller is a powerful nonlinear robust controller under condition of partly uncertain dynamic parameters of system [7]. This controller is used to control of highly nonlinear systems especially for robot manipulators. Chattering phenomenon and nonlinear equivalent dynamic formulation in uncertain dynamic parameter are two main drawbacks in pure sliding mode controller [20]. The main reason to opt for this controller is its acceptable control performance in wide range and solves two most important challenging topics in control which names, stability and robustness [7, 17-20]. Sliding mode controller is divided into two main sub controllers: discontinues controller(τ_{dts}) and equivalent controller(τ_{eq}). Discontinues controller causes an acceptable tracking performance at the expense of very fast switching. Conversely in this theory good trajectory following is based on fast switching, fast switching is caused to have system instability and chattering phenomenon. Fine tuning the sliding surface slope is based on nonlinear equivalent part [1, 6]. However, this controller is used in many applications but, pure sliding mode controller has two most important challenges: chattering phenomenon and nonlinear equivalent dynamic formulation in uncertain parameters[20]. Chattering phenomenon (Figure 1) can causes some problems such as saturation and heat the mechanical parts of robot manipulators or drivers. To reduce or eliminate the chattering, various papers have been reported by many researchers which classified into two most important methods: boundary layer saturation method and estimated uncertainties method [1]. In boundary layer saturation method, the basic idea is the discontinuous method replacement by saturation (linear) method with small neighborhood of the switching surface. This replacement caused to increase the error performance against with the considerable chattering reduction. Slotine and Sastry have introduced boundary layer method instead of discontinuous method to reduce the chattering[21]. Slotine has presented sliding mode with boundary layer to improve the industry application [22]. Palm has presented a fuzzy method to nonlinear approximation instead of linear approximation inside the boundary layer to improve the chattering and control the result performance[23]. Moreover, Weng and Yu improved the previous method by using a new method in fuzzy nonlinear approximation inside the boundary layer and adaptive method[24]. As mentioned [24]sliding mode fuzzy controller (SMFC) is fuzzy controller based on sliding mode technique to most exceptional stability and robustness. Sliding mode fuzzy controller has the two most important advantages: reduce the number of fuzzy rule base and increase robustness and stability.

Conversely sliding mode fuzzy controller has the above advantages, define the sliding surface slope coefficient very carefully is the main disadvantage of this controller. Estimated uncertainty method used in term of uncertainty estimator to compensation of the system uncertainties. It has been used to solve the chattering phenomenon and also nonlinear equivalent dynamic. If estimator has an acceptable performance to compensate the uncertainties, the chattering is reduced. Research on estimated uncertainty to reduce the chattering is significantly growing as their applications such as industrial automation and robot manipulator. For instance, the applications of artificial intelligence, neural networks and fuzzy logic on estimated uncertainty method have been reported in [25-28]. Wu et al. [30] have proposed a simple fuzzy estimator controller beside the discontinuous and equivalent control terms to reduce the chattering. Their design had three main parts i.e. equivalent, discontinuous and fuzzy estimator tuning part which has reduced the chattering very well. Elmali et al. [27] and Li and Xu [29] have addressed sliding mode control with perturbation estimation method (SMCPE) to reduce the classical sliding mode chattering. This method was tested for the tracking control of the first two links of a SCARA type HITACHI robot. In this technique, digital controller is used to increase the system's response quality. However this controller's response is very fast and robust but it has chattering phenomenon. Design a robust controller for robot manipulator is essential because robot manipulator has highly nonlinear dynamic parameters.

Computed torque controller (CTC) is a powerful nonlinear controller which it widely used in control of robot manipulator. It is based on feedback linearization and computes the required arm torques using the nonlinear feedback control law. This controller works very well when all dynamic and physical parameters are known but when the robot manipulator has variation in dynamic parameters, in this situation the controller has no acceptable performance[14]. In practice, most of physical systems (e.g., robot manipulators) parameters are unknown or time variant, therefore, computed torque like controller used to compensate dynamic equation of robot manipulator[1, 6]. Research on computed torque controller is significantly growing on robot manipulator application which has been reported in [1, 6, 15-16]. Vivas and Mosquera [15]have proposed a predictive functional controller and compare to computed torque controller for tracking response in uncertain environment. However both controllers have been used in feedback linearization, but predictive strategy gives better result as a performance. A computed torque control with non parametric regression models have been presented for a robot arm[16]. This controller also has been problem in uncertain dynamic models. Based on [1, 6]and [15-16] computed torque controller is a significant nonlinear controller to certain systems which it is based on feedback linearization and computes the required arm torques using the nonlinear feedback control law.

This paper is organized as follows:

In section 2, dynamic formulation of robot manipulator is presented. Detail of classical SMC and MATLAB/SIMULINK implementation of this controller is presented in section 3. Detail of classical CTC and MATLAB/SIMULINK implementation of this controller is presented in section 4. In section 5, the simulation result is presented and finally in section 6, the conclusion is presented.

2. PUMA 560 ROBOT MANIPULATOR DYNAMIC FORMULATION Dynamics of PUMA560 Robot Manipulator

To position control of robot manipulator, the second three axes are locked the dynamic equation of PUMA robot manipulator is given by [77-80];

$$M(\theta) \begin{bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{bmatrix} + B(\theta) \begin{bmatrix} \theta_1 \theta_2 \\ \theta_1 \theta_3 \\ \theta_2 \theta_3 \end{bmatrix} + C(\theta) \begin{bmatrix} \theta_1^2 \\ \theta_2^2 \\ \theta_3^2 \end{bmatrix} + G(\theta) = \begin{bmatrix} \tau_1 \\ \tau_2 \\ \tau_3 \end{bmatrix}$$
(1)

Where

$$M(q) = \begin{bmatrix} M_{11} & M_{12} & M_{13} & 0 & 0 & 0 \\ M_{21} & M_{22} & M_{23} & 0 & 0 & 0 \\ M_{31} & M_{32} & M_{33} & 0 & M_{35} & 0 \\ 0 & 0 & 0 & M_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & M_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & M_{66} \end{bmatrix}$$
 (2)

M is computed as

$$\begin{array}{l} M_{11} = & \text{o3} \\ I_{m1} + I_1 + I_3 \times \cos(\theta_2)\cos(\theta_2) + I_7\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3) + I_{10}\sin(\theta_2 + \theta_3)\cos(\theta_2) \\ I_{11}\sin(\theta_2)\cos(\theta_2) + I_{21}\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3) + 2 + [I_5\cos(\theta_2)\sin(\theta_2 + \theta_3) + I_{12}\cos(\theta_2)\cos(\theta_2 + \theta_3) + I_{15}\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3) + I_{16}\cos(\theta_2)\sin(\theta_2 + \theta_3) + I_{22} \\ \theta_3)\cos(\theta_2 + \theta_3) \end{array}$$

$$M_{12} = I_{4}\sin(\theta_{2}) + I_{8}\cos(\theta_{2} + \theta_{3}) + I_{9}\cos(\theta_{2}) + I_{13}\sin(\theta_{2} + \theta_{3}) - I_{18}\cos(\theta_{2} + \theta_{3})$$
 (4)

$$M_{13} = I_8 \cos(\theta_2 + \theta_3) + I_{13} \sin(\theta_2 + \theta_3) - I_{18} \cos(\theta_2 + \theta_3)$$
 (5)

$$M_{22} = I_{m2} + I_2 + I_6 + 2[I_5 \sin(\theta_3) + I_{12} \cos(\theta_2) + I_{15} + I_{16} \sin(\theta_3)]$$
 (6)

$$M_{23} = I_5 \sin(\theta_3) + I_6 + I_{12} \cos(\theta_3) + I_{16} \sin(\theta_3) + 2I_{15}$$
(7)

$$M_{33} = I_{m3} + I_6 + 2I_{15} \tag{8}$$

$$M_{35} = I_{15} + I_{17} \tag{9}$$

$$M_{44} = I_{1014} + I_{14} \tag{10}$$

$$M_{55} = I_{m5} + I_{17} \tag{11}$$

$$M_{66} = I_{23} + I_{23} \tag{12}$$

$$M_{21} = M_{12}$$
, $M_{31} = M_{13}$ and $M_{32} = M_{23}$ (13)

and Corilios (B) matrix is calculated as the following

Where,

$$b_{112} = 2[-I_3\sin(\theta_2)\cos(\theta_2) + I_5\cos(\theta_2 + \theta_2 + \theta_3) + I_7\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) - (15)\theta_2 + \theta_3) - I_{15}2\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) + I_{16}\cos(\theta_2 + \theta_2 + \theta_3) + I_{21}\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) + I_{22}(1 - 2\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3))] + I_{10}(1 - 2\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3)) + I_{23}\sin(\theta_2)\sin(\theta_2))$$

$$b_{113} = 2 \Big[I_5 \cos(\theta_2) \cos(\theta_2 + \theta_3) + I_7 \sin(\theta_2 + \theta_3) \cos(\theta_2 + \theta_3) - I_{12} \cos(\theta_2) \sin(\theta_2 - \theta_3) \Big] \\ I_{15} 2 \sin(\theta_2 + \theta_3) \cos(\theta_2 + \theta_3) + I_{16} \cos(\theta_2) \cos(\theta_2 + \theta_3) + I_{21} \sin(\theta_2 + \theta_3) \cos(\theta_2 + \theta_3) \Big] \\ I_{12} \Big(1 - 2 \sin(\theta_2 + \theta_3) \sin(\theta_2 + \theta_3) \Big) \Big] + I_{10} \Big(1 - 2 \sin(\theta_2 + \theta_3) \sin(\theta_2 + \theta_3) \Big)$$
 (16)

$$b_{115} = 2[-\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) + I_{15}2\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) + I_{16}\cos(\theta_2)\cos(\theta_3) + I_{12}\cos(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3)]$$
(17)

$$b_{123} = 2[-I_8\sin(\theta_2 + \theta_3) + I_{13}\cos(\theta_2 + \theta_3) + I_{18}\sin(\theta_2 + \theta_3)]$$
 (18)

$$b_{214} = I_{14}\sin(\theta_2 + \theta_3) + I_{19}\sin(\theta_2 + \theta_3) + 2I_{20}\sin(\theta_2 + \theta_3)(1 - 0.5)$$
(19)

$$b_{223} = 2[-I_{12}\sin(\theta_3) + I_{5}\cos(\theta_3) + I_{16}\cos(\theta_3)]$$
 (20)

$$b_{235} = 2[I_{16}\cos(\theta_3) + I_{22}] \tag{21}$$

$$b_{314} = 2[I_{20}\sin(\theta_2 + \theta_3)(1 - 0.5)] + I_{14}\sin(\theta_2 + \theta_3) + I_{19}\sin(\theta_2 + \theta_3)$$
 (22)

$$b_{412} = b_{214} = -[I_{14}\sin(\theta_2 + \theta_3) + I_{19}\sin(\theta_2 + \theta_3) + 2I_{20}\sin(\theta_2 + \theta_3)(1 - 0.5)]$$
 (23)

$$b_{413} = -b_{314} = -2[I_{20}\sin(\theta_2 + \theta_3)(1 - 0.5)] + I_{14}\sin(\theta_2 + \theta_3) + I_{19}\sin(\theta_2 + \theta_3)$$
 (24)

$$b_{415} = -I_{20}\sin(\theta_2 + \theta_3) - I_{17}\sin(\theta_2 + \theta_3)$$
 (25)

$$b_{514} = -b_{415} = I_{20}\sin(\theta_2 + \theta_3) + I_{17}\sin(\theta_2 + \theta_3)$$
 (26)

consequently coriolis matrix is shown as bellows;

$$B(q). q^{*}q^{*} = \begin{bmatrix} b_{112}. q_{1}q_{2} + b_{113}. q_{1}q_{3} + 0 + b_{123}. q_{2}q_{3} \\ 0 + b_{223}. q_{2}q_{3} + 0 + 0 \\ 0 \\ b_{412}. q_{1}q_{2} + b_{413}. q_{1}q_{3} + 0 \\ 0 \\ 0 \end{bmatrix}$$

$$(27)$$

Moreover Centrifugal (C) matrix is demonstrated as

Where,

$$c_{12} = I_4 \cos(\theta_2) - I_8 \sin(\theta_2 + \theta_3) - I_9 \sin(\theta_2) + I_{13} \cos(\theta_2 + \theta_3) + I_{16} \sin(\theta_2 + \theta_3)$$
 (29)

$$c_{13} = 0.5b_{123} = -I_8 \sin(\theta_2 + \theta_3) + I_{13} \cos(\theta_2 + \theta_3) + I_{18} \sin(\theta_2 + \theta_3)$$
(30)

$$\begin{array}{l} c_{21} = -0.5b_{112} = I_3\sin(\theta_2)\cos(\theta_2) - I_5\cos(\theta_2 + \theta_2 + \theta_3) - I_7\sin(\theta_2 + \theta_3)\cos(\theta_2 & \text{(31)} \\ I_{12}\sin(\theta_2 + \theta_2 + \theta_3) + I_{15}2\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) - I_{16}\cos(\theta_2 + \theta_2 + \theta_3) - I_{21}\sin(\theta_2 + \theta_3)\cos(\theta_2 + \theta_3) - I_{22}(1 - 2\sin(\theta_2 + \theta_3)\sin(\theta_2 + \theta_3)) - 0.5I_{10}(1 - 2\sin(\theta_2 + \theta_3)\sin(\theta_2)) \\ \theta_3)) - 0.5I_{11}(1 - 2\sin(\theta_2)\sin(\theta_2)) \end{array}$$

$$c_{22} = 0.5b_{223} = -I_{12}\sin(\theta_3) + I_5\cos(\theta_3) + I_{16}\cos(\theta_3)$$
(32)

$$\begin{array}{l} c_{23} = -0.5b_{113} = -I_{5}\cos(\theta_{2})\cos(\theta_{2}+\theta_{3}) - I_{7}\sin(\theta_{2}+\theta_{3})\cos(\theta_{2}+\theta_{3}) + I_{12}\cos(\theta_{2}+\theta_{3}) \\ \theta_{2}) - I_{15}2\sin(\theta_{2}+\theta_{3})\cos(\theta_{2}+\theta_{3}) - I_{16}\cos(\theta_{2})\cos(\theta_{2}+\theta_{3}) - I_{21}\sin(\theta_{2}+\theta_{3})\cos(\theta_{2}+\theta_{3}) \\ I_{22}\big(1-2\sin(\theta_{2}+\theta_{3})\sin(\theta_{2}+\theta_{3})\big) - 0.5I_{10}\big(1-2\sin(\theta_{2}+\theta_{3})\sin(\theta_{2}+\theta_{3})\big) \end{array} \tag{33}$$

$$c_{31} = -c_{23} = I_{12}\sin(\theta_3) - I_5\cos(\theta_3) - I_{16}\cos(\theta_3)$$
(34)

$$c_{32} = -0.5b_{115} = sin(\theta_2 + \theta_3)cos(\theta_2 + \theta_3) - I_{15}2sin(\theta_2 + \theta_3)cos(\theta_2 + \theta_3) - I_{16}cos(\theta_2)cos(\theta_2 + \theta_3)$$

$$I_{22}cos(\theta_2 + \theta_3)cos(\theta_2 + \theta_3)$$
(35)

$$c_{52} = -0.5b_{225} = -I_{16}\cos(\theta_3) - I_{22}$$
(36)

In this research $q_4 = q_5 = q_6 = 0$, as a result

$$C(q).q^{2} = \begin{bmatrix} c_{112}.q_{2}^{7} + c_{13}.q_{3}^{7} \\ c_{21}.q_{1}^{2} + c_{23}.q_{3}^{2} \\ c_{13}.q_{1}^{2} + c_{32}.q_{2}^{2} \\ 0 \\ c_{51}.q_{1}^{2} + c_{52}.q_{2}^{2} \end{bmatrix}$$
(37)

Gravity (G) Matrix can be written as

$$G(q) = \begin{bmatrix} 0 \\ g_2 \\ g_3 \\ 0 \\ g_5 \\ 0 \end{bmatrix}$$
(38)

Where.

$$G_2 = g_1 \cos(\theta_2) + g_2 \sin(\theta_2 + \theta_3) + g_3 \sin(\theta_2) + g_4 \cos(\theta_2 + \theta_3) + g_5 \sin(\theta_2 + \theta_3) + g_5 \sin(\theta_3 + \theta_3)$$

$$G_3 = g_2 \sin(\theta_2 + \theta_3) + g_4 \cos(\theta_2 + \theta_3) + g_5 \sin(\theta_2 + \theta_3)$$
(40)

$$G_5 = g_5 \sin \left(\theta_2 + \theta_3\right) \tag{41}$$

Suppose 4 is written as follows

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$$\ddot{q} = M^{-1}(q) \cdot \{ \tau - [B(q)\dot{q}\dot{q} + C(q)\dot{q}^2 + g(q)] \}$$
(42)

and K is introduced as

$$K = \{ \tau - [B(q)\dot{q}\dot{q} + C(q)\dot{q}^2 + g(q)] \}$$
(43)

acan be written as

$$\ddot{q} = M^{-1}(q). K \tag{44}$$

Therefore K for PUMA robot manipulator is calculated by the following equations

$$K_1 = \tau_1 - [b_{112}\dot{q}_1\dot{q}_2 + b_{113}\dot{q}_1\dot{q}_3 + 0 + b_{123}\dot{q}_2\dot{q}_3] - [c_{12}\dot{q}_2^2 + c_{13}\dot{q}_3^2] - g_1$$
(45)

$$K_2 = \tau_2 - [b_{223}\dot{q}_2\dot{q}_3] - [C_{21}\dot{q}_1^2 + C_{23}\dot{q}_3^2] - g_2$$
(46)

$$K_3 = \tau_3 - \left[C_{31}\dot{q}_1^2 + C_{32}\dot{q}_2^2 \right] - g_3 \tag{47}$$

$$K_4 = \tau_4 - [b_{412}\dot{q}_1\dot{q}_2 + b_{413}\dot{q}_1\dot{q}_3] - g_4 \tag{48}$$

$$K_{5} = \tau_{5} - [C_{51}\dot{q}_{1}^{2} + C_{52}\dot{q}_{2}^{2}] - g_{5}$$

$$\tag{49}$$

$$K_6 = \tau_6 \tag{50}$$

An information about inertial constant and gravitational constant are shown in Tables 1 and 2 based on the studies carried out by Armstrong [80] and Corke and Armstrong [81].

$I_1 = 1.43 \pm 0.05$	$I_2 = 1.75 \pm 0.07$
$I_3 = 1.38 \pm 0.05$	$I_4 = 0.69 \pm 0.02$
$I_5 = 0.372 \pm 0.031$	$I_6 = 0.333 \pm 0.016$
$I_7 = 0.298 \pm 0.029$	$I_s = -0.134 \pm 0.014$
$I_9 = 0.0238 \pm 0.012$	$I_{10} = -0.0213 \pm 0.0022$
$I_{11} = -0.0142 \pm 0.0070$	$I_{12} = -0.011 \pm 0.0011$
$I_{13} = -0.00379 \pm 0.0009$	$I_{14} = 0.00164 \pm 0.000070$
$I_{15} = 0.00125 \pm 0.0003$	$I_{16} = 0.00124 \pm 0.0003$
$I_{17} = 0.000642 \pm 0.0003$	$I_{18} = 0.000431 \pm 0.00013$
$I_{19} = 0.0003 \pm 0.0014$	$I_{20} = -0.000202 \pm 0.0008$
$I_{21} = -0.0001 \pm 0.0006$	$I_{22} = -0.000058 \pm 0.00001$
$I_{23} = 0.00004 \pm 0.00002$	$I_{\rm m1} = 1.14 \pm 0.27$
$I_{m2} = 4.71 \pm 0.54$	$I_{ms} = 0.827 \pm 0.093$
$I_{m4} = 0.2 \pm 0.016$	$I_{\text{ms}} = 0.179 \pm 0.014$
$I_{m6} = 0.193 \pm 0.016$	

TABLE 1: Inertial constant reference (*Kg.m*²)

$g_1 = -37.2 \pm 0.5$	$g_2 = -8.44 \pm 0.20$
$g_3 = 1.02 \pm 0.50$	$g_4 = 0.249 \pm 0.025$
$g_5 = -0.0282 \pm 0.0056$	

TABLE 2: Gravitational constant (*N.m*)

3. CONTROL: SLIDING MODE CONTROLLER ANALYSIS, MODELLING AND IMPLEMENTATION ON PUMA 560 ROBOT MANIPULATOR

In this section formulations of sliding mode controller for robot manipulator is presented based on [1, 6]. Consider a nonlinear single input dynamic system is defined by [6]: $\mathbf{x}^{(n)} = f(\vec{x}) + b(\vec{x})u$ (51)

Where u is the vector of control input, $\mathbf{x}^{(n)}$ is the \mathbf{n}^{th} derivation of \mathbf{x} , $\mathbf{x} = [\mathbf{x}, \dot{\mathbf{x}}, \ddot{\mathbf{x}}, \dots, \mathbf{x}^{(n-1)}]^T$ is the state vector, $\mathbf{f}(\mathbf{x})$ is unknown or uncertainty, and $\mathbf{b}(\mathbf{x})$ is of known sign function. The main goal to design this controller is train to the desired state; $\mathbf{x}_d = [\mathbf{x}_d, \dot{\mathbf{x}}_d, \ddot{\mathbf{x}}_d, \dots, \mathbf{x}_d]^{(n-1)}]^T$, and trucking error vector is defined by [6]:

$$\widetilde{\mathbf{x}} = \mathbf{x} - \mathbf{x}_d = [\widetilde{\mathbf{x}}, \dots, \widetilde{\mathbf{x}}^{(n-1)}]^T \tag{52}$$

A time-varying sliding surface s(x,t) in the state space R^n is given by [6]:

$$s(x,t) = \left(\frac{d}{dt} + \lambda\right)^{n-1} \tilde{x} = 0$$
 (53)

where λ is the positive constant. To further penalize tracking error, integral part can be used in

sliding surface part as follows [6]:

$$s(x,t) = (\frac{d}{dt} + \lambda)^{n-1} \left(\int_0^t \tilde{x} dt \right) = 0$$
(54)

The main target in this methodology is kept the sliding surface slope $\mathfrak{s}(x,\mathfrak{t})$ near to the zero.

Therefore, one of the common strategies is to find input
$$U$$
 outside of $s(x,t)$ [6].
$$\frac{1}{2} \frac{d}{dt} s^2(x,t) \le -\zeta |s(x,t)|$$
 (55)

where ζ is positive constant.

If
$$S(0)>0 \rightarrow \frac{d}{dt}S(t) \leq -\zeta$$
 (56)

To eliminate the derivative term, it is used an integral term from t=0 to $t=t_{reach}$

$$\int_{t=0}^{t=t_{reach}} \frac{d}{dt} S(t) \le -\int_{t=0}^{t=t_{reach}} \eta \to S(t_{reach}) - S(0) \le -\zeta(t_{reach} - 0)$$
(57)

Where t_{reach} is the time that trajectories reach to the sliding surface so, suppose $S(t_{reach} = 0)$ defined as

$$0 - S(0) \le -\eta(t_{reach}) \to t_{reach} \le \frac{S(0)}{\zeta}$$
(58)

and

$$if S(0) < 0 \rightarrow 0 - S(0) \leq -\eta(t_{reach}) \rightarrow S(0) \leq -\zeta(t_{reach}) \rightarrow t_{reach} \leq \frac{|S(0)|}{\eta}$$
(59)

Equation (81) guarantees time to reach the sliding surface is smaller than $\frac{|S(0)|}{\epsilon}$ since the trajectories are outside of S(t).

$$if S_{treeh} = S(0) \rightarrow error(x - x_d) = 0$$
(60)

suppose S is defined as

$$s(x,t) = (\frac{d}{dt} + \lambda) \quad \tilde{x} = (\dot{x} - \dot{x}_d) + \lambda(x - x_d)$$
(61)

The derivation of S, namely, \$\frac{s}{2}\$ can be calculated as the following;

$$S = (\ddot{\mathbf{x}} - \ddot{\mathbf{x}}_{d}) + \lambda(\ddot{\mathbf{x}} - \dot{\mathbf{x}}_{d}) \tag{62}$$

suppose the second order system is defined as;

$$\ddot{x} = f + u \rightarrow \dot{S} = f + U - \ddot{x}_d + \lambda(\dot{x} - \dot{x}_d) \tag{63}$$

Where f is the dynamic uncertain, and also since S = 0 and S = 0, to have the best approximation , is defined as

$$\hat{U} = -\hat{f} + \ddot{x}_d - \lambda(\dot{x} - \dot{x}_d) \tag{64}$$

A simple solution to get the sliding condition when the dynamic parameters have uncertainty is the switching control law:

$$U_{dis} = \hat{U} - K(\vec{x}, t) \cdot \text{sgn}(s)$$
(65)

where the switching function sgn (5) is defined as [1, 6]

$$sgn(s) = \begin{cases} 1 & s > 0 \\ -1 & s < 0 \\ 0 & s = 0 \end{cases}$$
 and the $K(\vec{x},t)$ is the positive constant. Suppose by (67) the following equation can be written as,

$$\frac{1}{2}\frac{d}{dt}s^{2}(x,t) = S \cdot S = \left[f - \hat{f} - K \operatorname{sgn}(s)\right] \cdot S = \left(f - \hat{f}\right) \cdot S - K|S|$$
(67)

and if the equation (61) instead of (60) the sliding surface can be calculated as

$$s(x,t) = \left(\frac{d}{dt} + \lambda\right)^2 \left(\int_0^t \tilde{x} \, dt\right) = (\dot{x} - \dot{x}_d) + 2\lambda(\dot{x} - \dot{x}_d) - \lambda^2(x - x_d)$$
(68)

in this method the approximation of \boldsymbol{U} is computed as [6]

$$\widehat{U} = -\widehat{f} + \ddot{x}_d - 2\lambda(\dot{x} - \dot{x}_d) + \lambda^2(x - x_d)$$
(69)

Based on above discussion, the sliding mode control law for a multi degrees of freedom robot manipulator is written as [1, 6]:

$$\tau = \tau_{ea} + \tau_{dis} \tag{70}$$

Where, the model-based component au_{eq} is the nominal dynamics of systems and au_{eq} for first 3 DOF PUMA robot manipulator can be calculate as follows [1]:

$$\tau_{\epsilon q} = [M^{-1}(B + C + G) + S]M \tag{71}$$

and τ_{dis} is computed as [1]; $\tau_{dis} = K \cdot \operatorname{sgn}(S)$

$$\tau_{dis} = K \cdot \operatorname{sgn}(S) \tag{72}$$

by replace the formulation (72) in (70) the control output can be written as;

$$\tau = \tau_{eq} + K.\operatorname{sgn}(S) \tag{73}$$

Figure 1 shows the position classical sliding mode control for PUMA 560 robot manipulator. By (73) and (71) the sliding mode control of PUMA 560 robot manipulator is calculated as;

$$\tau = [M^{-1}(B+C+G)+S]M+K \cdot \operatorname{sgn}(S)$$
(74)

where $S = \lambda e + \dot{e}$ in PD-SMC and $S = \lambda e + \dot{e} + (\frac{\lambda}{a})^2 \sum e$ in PID-SMC.

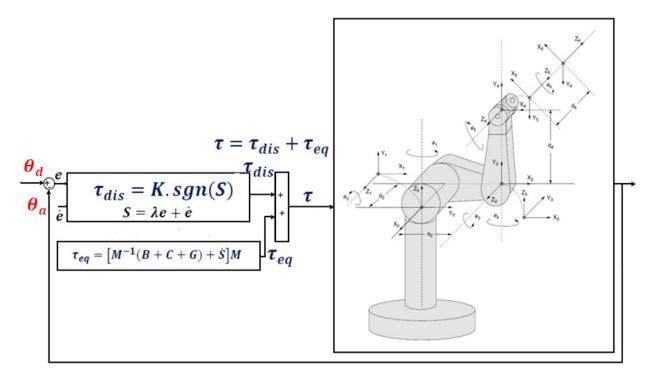


FIGURE 1: Block diagram of pure sliding mode controller with switching function

Implemented Sliding Mode Controller

The main object is implementation of controller block. According to T _{dis} equation which is T _{dis}=K * sign(s), this part will be created like figure 2. As it is obvious, the parameter e is the difference of actual and desired values and # is the change of error. Luanda (I1) and k are coefficients which are affected on discontinuous component and the saturation function accomplish the switching progress. A sample of discontinuous torque for one joint is like Figure 2.

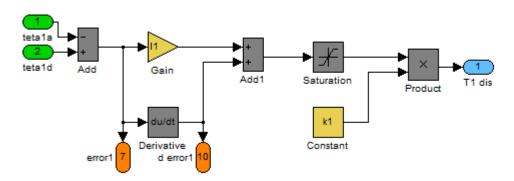


FIGURE 2: Discontinuous part of torque for one joint variable

As it is seen in figure 2 the error value and the change of error were chosen to exhibit in measurement center. In this block by changing gain and coefficient values, the best control system will be applied. In the second step according to torque formulation in SMC mode, the equivalent part should constructed. Based on equivalent formulation $\tau_{\varepsilon q} = [M^{-1}(B + C + G) + 5]M$ all constructed blocks just connect to each other as Figure 3. In this figure the N (q, \dot{q}) is the dynamic parameters block (i.e., A set of Coriolis, Centrifugal and

Gravity blocks) and the derivative of S is apparent. Just by multiplication and summation, the output which is equivalent torque will be obtained.

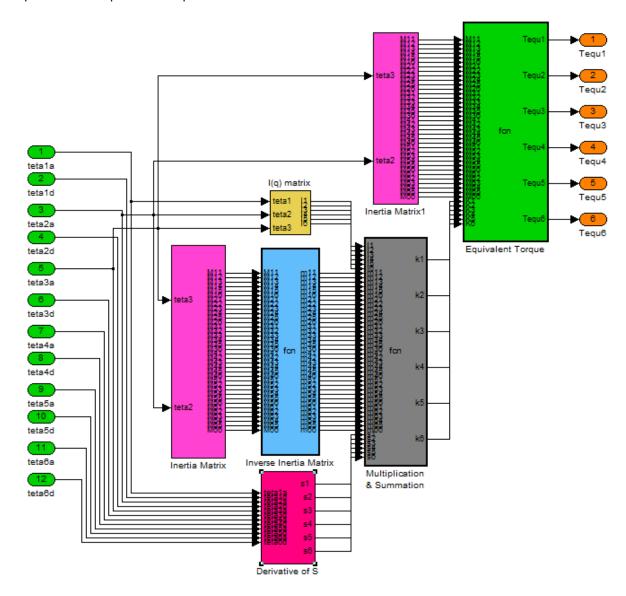


FIGURE 3: The equivalent part of torque with required blocks

The inputs are thetas and the final outputs are equivalent torque values. The relations between other blocks are just multiplication and summation as mentioned in torque equation. The next phase is calculation of the summation of equivalent part and discontinuous part which make the total torque value. This procedure is depicted in Figure 4.

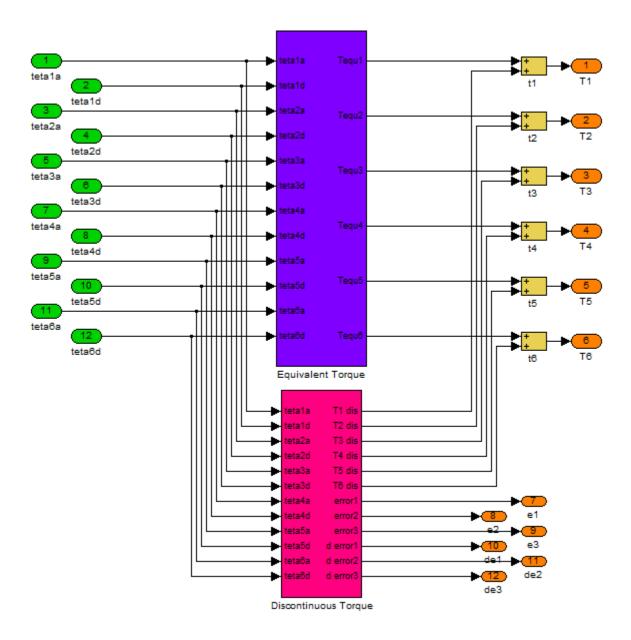


FIGURE 4: The total value of torque which is summation of equivalent & discontinuous blocks

In the next step transform our subsystems into a general system to form controller block and the outputs will be connected to the plant, in order to execute controlling process. Then, trigger the main inputs with power supply to check validity and performance. In Figure 5 Dynamics, Kinematics, Controller and the measurement center blocks are shown.

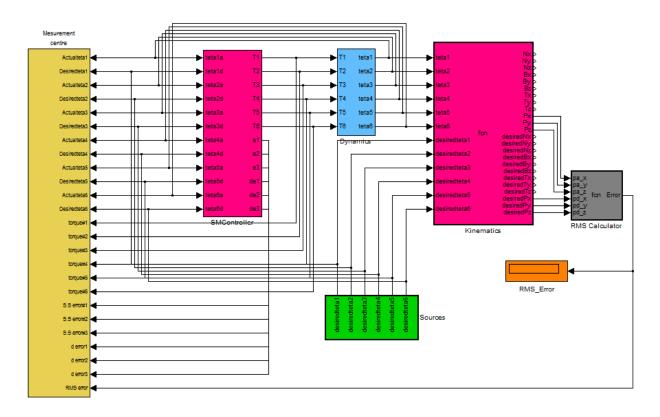


FIGURE 5: Measurement center, Controller, Dynamics and Kinematics Blocks

4. CONTROL: COMPUTED TORQUE CONTROLLER ANALYSIS, MODELLING AND IMPLEMENTATION ON PUMA 560 ROBOT MANIPULATOR

Computed torque controller (CTC) is a powerful nonlinear controller which it widely used in control of robot manipulator. It is based on feedback linearization and computes the required arm torques using the nonlinear feedback control law. This controller works very well when all dynamic and physical parameters are known but when the robot manipulator has variation in dynamic parameters, in this situation the controller has no acceptable performance[14]. In practice, most of physical systems (e.g., robot manipulators) parameters are unknown or time variant, therefore, computed torque like controller used to compensate dynamic equation of robot manipulator[1, 6]. Research on computed torque controller is significantly growing on robot manipulator application which has been reported in [1, 6, 15-16]. Vivas and Mosquera [15]have proposed a predictive functional controller and compare to computed torque controller for tracking response in uncertain environment. However both controllers have been used in feedback linearization, but predictive strategy gives better result as a performance. A computed torque control with non parametric regression models have been presented for a robot arm[16]. This controller also has been problem in uncertain dynamic models. Based on [1, 6]and [15-16] computed torque controller is a significant nonlinear controller to certain systems which it is based on feedback linearization and computes the required arm torques using the nonlinear feedback control law. When all dynamic and physical parameters are known, computed torque controller works fantastically; practically a large amount of systems have uncertainties, therefore sliding mode controller is one of the best case to solve this challenge.

The central idea of Computed torque controller (CTC) is feedback linearization so, originally this algorithm is called feedback linearization controller. It has assumed that the desired motion

trajectory for the manipulator $q_d(t)$, as determined, by a path planner. Defines the tracking error as:

$$e(t) = q_d(t) - q_a(t) \tag{75}$$

Where e(t) is error of the plant, $q_d(t)$ is desired input variable, that in our system is desired displacement, $q_a(t)$ is actual displacement. If an alternative linear state-space equation in the form $\dot{x} = Ax + BU$ can be defined as

form
$$\dot{x} = Ax + BU$$
 can be defined as
$$\dot{x} = \begin{bmatrix} 0 & I \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ I \end{bmatrix} U$$
(76)

With $U = -M^{-1}(q) \cdot N(q, \dot{q}) + M^{-1}(q) \cdot \tau$ and this is known as the Brunousky canonical form. By equation (76) and (77) the Brunousky canonical form can be written in terms of the state $\mathbf{x} = [\mathbf{e}^T \dot{\mathbf{e}}^T]^T$ as [1]:

$$\mathbf{x} = \begin{bmatrix} \mathbf{e}^T \ \mathbf{e}^T \end{bmatrix}^T \text{ as } \begin{bmatrix} 1 \end{bmatrix}: \\
\frac{d}{dt} \begin{bmatrix} \mathbf{e} \\ \mathbf{e} \end{bmatrix} = \begin{bmatrix} 0 & I \\ 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} \mathbf{e} \\ \mathbf{e} \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} U$$
(77)

With

$$U = \ddot{q}_d + M^{-1}(q), \{N(q, \dot{q}) - \tau\}$$
(78)

Then compute the required arm torques using inverse of equation (79), is;

$$\tau = M(q)(\dot{q}_d - U) + N(\dot{q}, q) \tag{79}$$

This is a nonlinear feedback control law that guarantees tracking of desired trajectory. Selecting proportional-plus-derivative (PD) feedback for U(t) results in the PD-computed torque controller [6];

$$\tau = M(q) \left(\ddot{q}_d + K_v \dot{e} + K_u e \right) + N(q, \dot{q}) \tag{80}$$

and the resulting linear error dynamics are

$$(\ddot{q}_d + K_v \dot{e} + K_v e) = 0$$
 (81)

According to the linear system theory, convergence of the tracking error to zero is guaranteed [6]. Where \mathbf{K}_p and \mathbf{K}_r are the controller gains. The result schemes is shown in Figure 6, in which two feedback loops, namely, inner loop and outer loop, which an inner loop is a compensate loop and an outer loop is a tracking error loop.

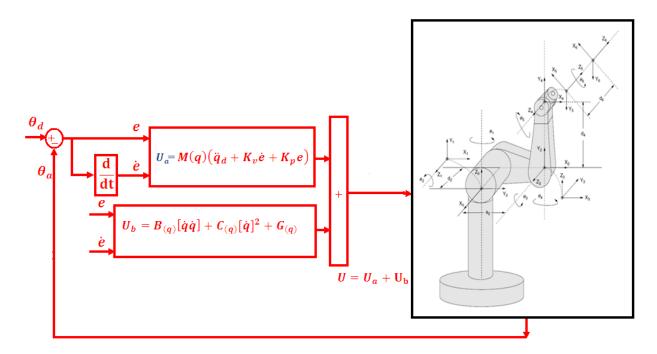


FIGURE 6: Block diagram of PD-computed torque controller (PD-CTC)

The application of proportional-plus-derivative (PD) computed torque controller to control of PUMA 560 robot manipulator introduced in this part. Suppose that in (82) the nonlinearity term defined by the following term

$$N(q, \dot{q}) = B(q)\dot{q}\dot{q} + C(q)\dot{q}^{2} + g(q)$$

$$\begin{bmatrix} b_{112}\dot{q}_{1}\dot{q}_{2} + b_{113}\dot{q}_{1}\dot{q}_{3} + 0 + b_{123}\dot{q}_{2}\dot{q}_{3} \\ 0 + b_{223}\dot{q}_{2}\dot{q}_{3} + 0 + 0 \\ 0 \\ b_{412}\dot{q}_{1}\dot{q}_{2} + b_{413}\dot{q}_{1}\dot{q}_{3} + 0 + 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} C_{12}\dot{q}_{2}^{2} + C_{13}\dot{q}_{3}^{2} \\ C_{21}\dot{q}_{1}^{2} + C_{23}\dot{q}_{3}^{2} \\ C_{31}\dot{q}_{1}^{2} + C_{32}\dot{q}_{2}^{2} \end{bmatrix} + \begin{bmatrix} 0 \\ g_{2} \\ g_{3} \\ 0 \\ g_{5} \\ 0 \end{bmatrix}$$

$$(82)$$

Therefore the equation of PD-CTC for control of PUMA 560 robot manipulator is written as the equation of (83);

$$\begin{bmatrix} \widehat{\tau}_{1} \\ \widehat{\tau}_{2} \\ \widehat{\tau}_{3} \\ \widehat{\tau}_{4} \\ \widehat{\tau}_{5} \\ \widehat{\tau}_{6} \end{bmatrix} = \begin{bmatrix} M_{11} & M_{12} & M_{13} & 0 & 0 & 0 \\ M_{21} & M_{22} & M_{23} & 0 & 0 & 0 \\ M_{31} & M_{32} & M_{33} & 0 & M_{35} & 0 \\ 0 & 0 & 0 & M_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & M_{55} & 0 \\ 0 & 0 & 0 & 0 & M_{55} & 0 \\ 0 & 0 & 0 & 0 & M_{66} \end{bmatrix} \begin{bmatrix} \ddot{q}_{d1} + K_{v1}\dot{e}_{1} + K_{p1}e_{1} \\ \ddot{q}_{d2} + K_{v2}\dot{e}_{2} + K_{p2}e_{2} \\ \ddot{q}_{d3} + K_{v3}\dot{e}_{3} + K_{p3}e_{3} \\ \ddot{q}_{d4} + K_{v4}\dot{e}_{4} + K_{p4}e_{4} \\ \ddot{q}_{d5} + K_{v5}\dot{e}_{5} + K_{p5}e_{5} \\ \ddot{q}_{d6} + K_{v6}\dot{e}_{6} + K_{p6}e_{6} \end{bmatrix}$$

$$+ \begin{bmatrix} b_{112}\dot{q}_{1}\dot{q}_{2} + b_{113}\dot{q}_{1}\dot{q}_{3} + 0 + b_{123}\dot{q}_{2}\dot{q}_{3} \\ 0 + b_{223}\dot{q}_{2}\dot{q}_{3} + 0 + 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} C_{12}\dot{q}_{2}^{2} + C_{13}\dot{q}_{3}^{2} \\ C_{21}\dot{q}_{1}^{2} + C_{22}\dot{q}_{2}^{2} \\ C_{31}\dot{q}_{1}^{2} + C_{32}\dot{q}_{2}^{2} \\ 0 \\ C_{51}\dot{q}_{1}^{2} + C_{52}\dot{q}_{2}^{2} \end{bmatrix} + \begin{bmatrix} 0 \\ g_{2} \\ g_{3} \\ 0 \\ g_{5} \\ 0 \end{bmatrix}$$

The controller based on a formulation (83) is related to robot dynamics therefore it has problems in uncertain conditions.

Implemented Computed Torque Controller

In first step, constructed dynamics and kinematics blocks (i.e., plant) with power supply will be put in work space. The main object is implementation of controller block. According to PD equation which is $\ddot{q}_{cl} + K_{v} \dot{e}_{1} + K_{v} e$, the linearized part will be created like Figure 7. The linearized part so called PID. As it is obvious, the parameter e is the difference of actual and desired values and \ddot{e} is the change of error. K $_{p}$ and k $_{v}$ are proportional and derivative gains and \ddot{q}_{d} is double derivative of the joint variable. A sample of PD controller block for one joint is like Figure 8.

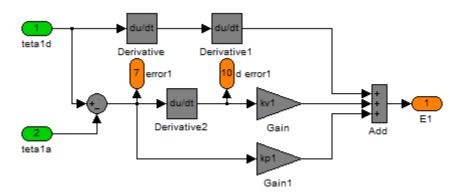


FIGURE 7: PD Controller for a joint variable

As it is seen in Figure 7 the error value and the change of error were chosen to exhibit in measurement center. In this block by changing gain values, the best control system will be applied. In the second step according to torque formulation in CTC mode, all constructed blocks just connect to each other as blew. In Figure 8 the N ($q, \frac{1}{4}$) is the dynamic parameters block (i.e., A set of Coriolis, Centrifugal and Gravity blocks).

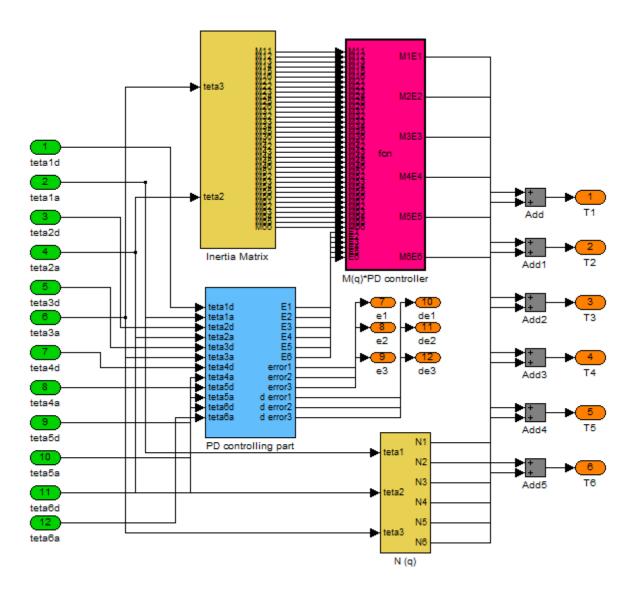


FIGURE 8: The general diagram of controller

The inputs are thetas and the final outputs are torque values. The relations between other blocks are just multiplication and summation as mentioned in torque equation. In the next step transform our subsystems into a general system to form controller block and the outputs will be connected to the plant, in order to execute controlling process. Then, trigger the main inputs with power supply to check validity and performance. In Figure 9, Dynamics, Kinematics and Controller blocks are shown.

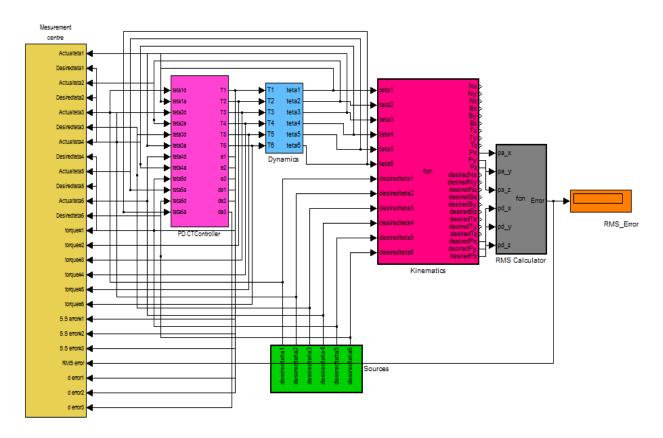


FIGURE 9: Controller, Dynamics and Kinematics Blocks

5. RESULTS

Sliding mode controller (SMC) and computed torque controller (CTC) were tested to Step responses. In this simulation the first, second, and third joints are moved from home to final position without and with external disturbance. The simulation was implemented in MATLAB/SIMULINK environment. It is noted that, these systems are tested by band limited white noise with a predefined 40% of relative to the input signal amplitude which the sample time is equal to 0.1. This type of noise is used to external disturbance in continuous and hybrid systems.

Tracking Performances

based on (90) in computed torque controller; the performance of this controller is depended on the PD (K_p and K_v) coefficients. These coefficients are $K_p = 30$ and $K_v = 4$ and computed by trial and error. Table 3 shows the different coefficient gain in computed torque controller.

	k_{P_1}	k_{V_1}	k_{p_2}	k_{V_2}	k_{p_3}	k_{V_3}	RMs	SS	SS	SS error ₃
	-1	.1	P2	7.2	P 3	7.3	error	error ₁	error ₂	J
1	8	4	8	4	8	4	2.276e-5	-3.81e-5	-3.81e-5	-3.81e-5
2	30	4	30	4	30	4	1.34e-5	-3.6e-5	-2.54e-5	-1.6e-5
3	1	4	1	4	1	4	0.0039	0.0065	0.0065	0.0065
4	8	40	8	40	8	40	0.502	5.043	5.043	5.043
5	8	0.5	8	0.5	8	0.5	0.0026	0.0043	0.0043	0.0043

TABLE 3: Tuning parameters of a step PD-CTC

Based on this table, the different PD coefficient gain has the different errors therefore minimum error played important role to select the $K_D = 30$ and $K_V = 4$.

Based on (80) in sliding mode controller with switching function; the performance is depended on the gain updating factor (K) and sliding surface slope coefficient (A). These two coefficients are computed by trial and error based on Table 4.

	$\lambda_{_{1}}$	$k_{\scriptscriptstyle 1}$	λ_2	k_2	λ_3	k_3	SS	SS	SS	RMS
	1	1	2	2		,	error ₁	error ₂	error ₃	error
data 1	3	30	6	30	6	30	0.1e-3	0.1e-3	-5.3e- 15	0.1e-4
data 2	3 0	30	60	30	6 0	30	-5.17	14.27	-1.142	0.05
data 3	3	30 0	6	30 0	6	30 0	2.28	0.97	0.076	0.08

TABLE 4: Tuning parameters of Step SMC with switching function

Based on Table 4, in this research for step inputs $\lambda_1=3$ and $\lambda_2=\lambda_3=6$; $K_1=K_2=K_3=30$ and for ramp inputs $\lambda_1=\lambda_2=15$ and $\lambda_2=10$; $K_1=K_2=K_3=5$.

Based on (84) in sliding mode controller with saturation function (boundary layer); the performance is depended on the gain updating factor (\mathbb{K}), sliding surface slope coefficient (\mathbb{A}) and boundary layer saturation coefficient (\mathbb{A}). These three coefficients are computed by trial and error based on Tables 5.

	$k_{\scriptscriptstyle 1}$	$\lambda_{_{1}}$	$\phi_{_{1}}$	k_{2}	λ_{2}	ϕ_2	k_3	$\lambda_{_3}$	ϕ_3	SS	SS	SS	RMS
	1	1	, 1	2	2	. 4	,	,		error ₁	error ₂	error ₃	error
dat	10	1	0.1	10	6	0.1	10	8	0.1	1e-6	1e-6	1e-6	1.2e-6
a1													
dat	10	1	0.1	10	6	0.1	10	8	0.1	0.2	0.05	-0.02	-0.037
a2	0			0			0						
dat	10	10	0.1	10	60	0.1	10	80	0.1	0.22	-0.21	-0.19	0.09
а3													

TABLE 5: Tuning parameters of a step SMC with boundary layer

In this research based on Table 5, for step inputs: $\lambda_1 = 1$, $\lambda_2 = 6$, $\lambda_3 = 8$; $K_1 = K_2 = K_3 = 10$; $\emptyset_1 = \emptyset_2 = \emptyset_3 = 0.1$ and for ramp inputs: $\lambda_1 = \lambda_2 = 15$, $\lambda_3 = 10$; $K_1 = K_2 = K_3 = 5$; $\emptyset_1 = \emptyset_2 = \emptyset_3 = 0.1$. Figure 10 and 4.2 is shown tracking performance for CTC, SMC with switching function and SMC with boundary layer without disturbance for step trajectorie.

Based on Figure 10; by comparing step response trajectory without disturbance in CTC, SMC with switching function and SMC with boundary layer, SMC with saturation function's overshoot about (0.94%) is lower than SMC with switching function's and CTC's (6.44%). SMC with switching function's and CTC's rise time (0.403) is lower than SMC with saturation function's (0.483). Based on simulation results the Steady State and RMS error in SMC with boundary layer (Steady State error =1e-6 and RMS error=1.2e-6) are fairly lower than CTC's (Steady State error $\cong -3e - 5$ and RMS error=-1.34e - 5) and SMC with switching function's (Steady State error $\cong -0.001$ and RMS error=0.00652). Based on Figure 10 and chattering phenomenon challenge in pure sliding mode controller with switching function, it is found fairly fluctuations in trajectory responses.

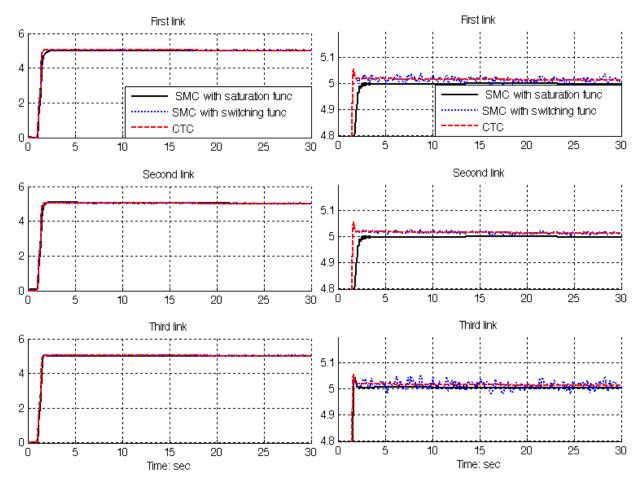


FIGURE 10: SMC with boundary layer, SMC with switching function and CTC for first, second and third joints: step trajectory

Based on Figure 10, step trajectory performance is used for comparisons of above controllers in certain systems. In this state CTC and SMC with saturation function has an acceptable trajectory performance but SMC with switching function has oscillation.

Disturbance rejection: Figures 11, 12 and 13 shows the CTC, SMC with switching function and SMC with saturation function (boundary layer).

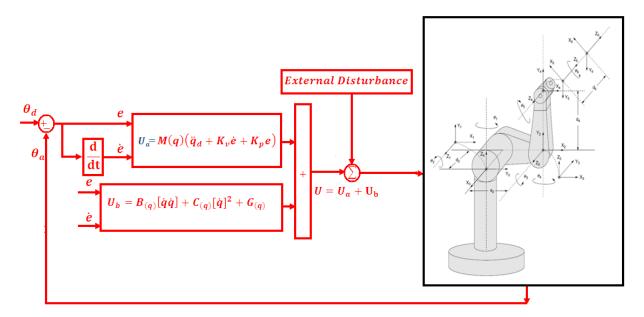


FIGURE 11: Computed torque controller (CTC) with disturbance

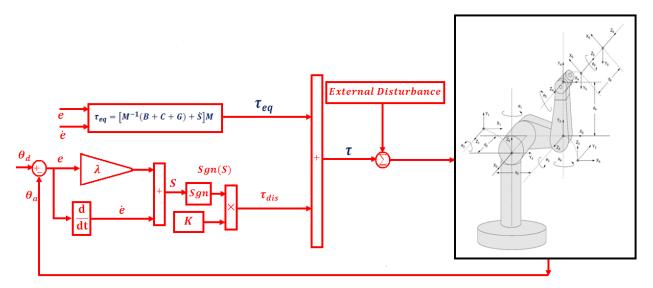


FIGURE 12: Sliding mode controllers with switching function with disturbance

Figure 14 shows the power disturbance elimination in CTC, SMC with switching function and SMC with boundary layer without disturbance for step trajectory. The disturbance rejection is used to test the robustness comparisons of these three controllers for step trajectory. A band limited white noise with predefined of 40% the power of input signal is applied to the step trajectory. It found fairly fluctuations in trajectory responses.

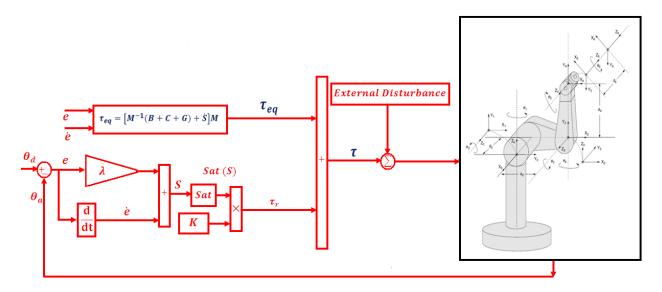


FIGURE 13: Sliding mode controllers with saturation function with disturbance

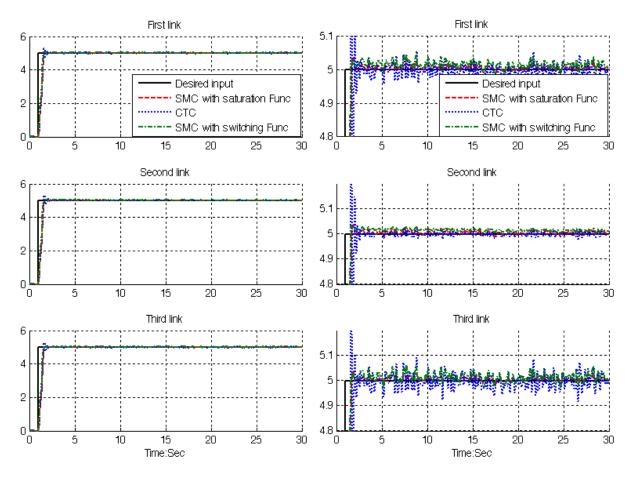


FIGURE 14: Desired input, SMC with boundary layer, SMC with switching function and CTC for first, second and third joints with 40% external disturbance: step trajectory

Based on Figure 14; by comparing step response trajectory with 40% disturbance of relative to the input signal amplitude in CTC, boundary layer (saturation) SMC and switching mode SMC. SMC with saturation function's overshoot about (6%) is lower than SMC with switching function's (13%) and CTC's (14.8%). SMC with switching function's and CTC's rise time (0.5) is lower than SMC with saturation function's (0.53). Besides the Steady State and RMS error in SMC with boundary layer (Steady State error =10e-4 and RMS error=11e-4) are fairly lower than CTC's (Steady State error ≅ 0.005 and RMS error=0.075) and SMC with switching function's (Steady State error $\cong -0.98$ and RMS error=0.98). Based on Figure 14, all three controllers have oscillation in trajectory response with regard to 40% of the input signal amplitude disturbance. All these three controllers have fairly large oscillation with regard to the external disturbance. Based on Figure 14, a step trajectory performance is used for comparisons of above controllers in presence of uncertainties with 40% of input signal amplitude. In this state however boundary layer SMC has relatively moderate oscillations but it is more robust than CTC and SMC with switching function. From the disturbance rejection for boundary layer SMC, SMC with switching function and CTC in presence of disturbance, it was seen that however boundary layer SMC performance is better than SMC with switching function and CTC but it has a limitation against to highly external disturbance.

Steady State and RMS Error

Figures 15and 16 show the error performance without disturbance and in presence of disturbance in CTC, SMC with switching function and SMC with boundary layer without disturbance for two type trajectories. The error performance is used to test the disturbance effect comparisons of these three controllers for step trajectory. A band limited white noise with predefined of 40% the power of input signal is applied to the step trajectory. It found fairly fluctuations in error responses.

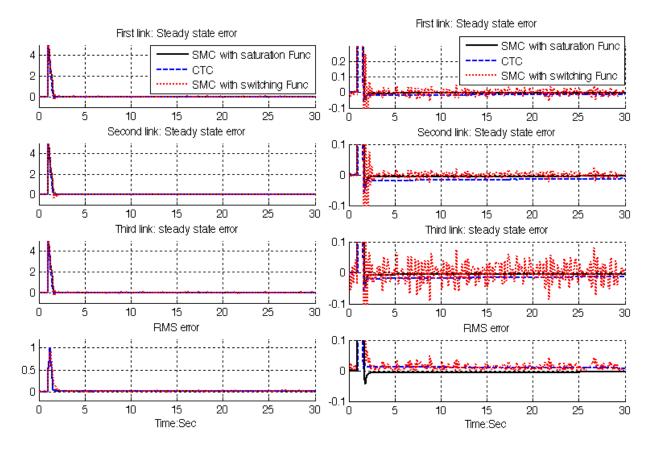


FIGURE 15: SMC with boundary layer, SMC with switching function and CTC for first, second and third joints steady state and RMS error without disturbance: step trajectory

Figure 15 shows error performance for first three links PUMA robot manipulator in CTC, SMC with switching function and SMC with boundary layer without disturbance for step trajectory. Based on Figure 10, all three joint's inputs are step function with the same step time (step time= 1 second), the same initial value (initial value=0) and the same final value (final value=5). Based on Figure 10, all three controllers have about the same rise time (rise time=0.4 second) which it is caused to create a needle wave in the range of 5 (amplitude=5) and the time width of 0.6 second. In this system this time is transient time and this part of error is transient error. The SMC with boundary layer and CTC give considerable error performance when compared to SMC with switching function. Besides the Steady State and RMS error in SMC with boundary layer (Steady State error =1e-6 and RMS error=1.2e-6) are fairly lower than CTC's (Steady State error $\cong -3\varepsilon - 5$ and RMS error=0.00652). Based on literature [17-18, 20-21] about chattering phenomenon, sliding mode controller with switching function has fairly oscillation in this system without any disturbance.

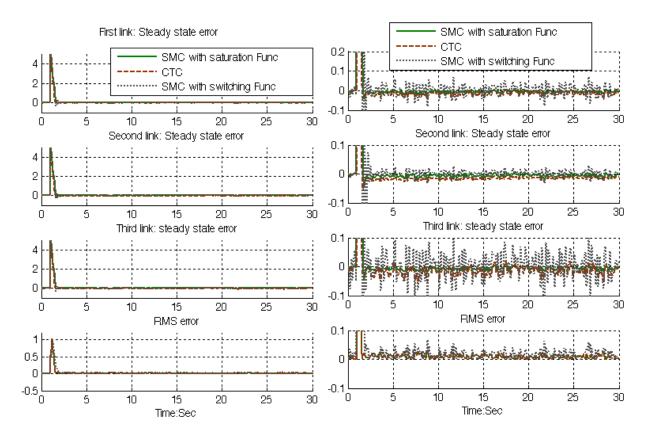


FIGURE 16: SMC with boundary layer, SMC with switching function and CTC First, second and third link steady state and RMS error with disturbance: step trajectory

Figure 16 shows steady state and RMS error performance for first three links PUMA robot manipulator in CTC, SMC with switching function and SMC with boundary layer with 40% disturbance for step trajectory. Based on Figure 10, all three joint's inputs are step function with the same step time (step time= 1 second), the same initial value (initial value=0) and the same final value (final value=5). Based on Figure 10, all three controllers have about the same rise time (rise time=0.4 second) which it is caused to create a needle wave in the range of 5 (amplitude=5) and the time width of 0.6 second. In this system this time is transient time and this part of error is transient error. The SMC with boundary layer gives considerable error performance when compared to CTC and SMC with switching function. Besides the Steady State and RMS error in SMC with boundary layer (Steady State error =10e-5 and RMS error=11e-4) are fairly lower than CTC's (Steady State error ≅ 0.005 and RMS error=0.075) and SMC with switching function's (Steady State error ≅ -0.98 and RMS error=0.98). Based on Figure 16, all three controllers have oscillation in error response with regard to 40% of the input signal amplitude disturbance. When applied 40% disturbances in SMC with boundary layer the RMS error increased approximately 9.17% (percent of increase the steady state (40% disturbance RMS ervor CTC the **RMS** error increased = 9.17%and in no disturbance AMS error 56% (percent of increase the approximately steady state error= (40% distur nce RMS error 0.075 no disturbance RMS error

Chattering Phenomenon

Chattering is one of the most important challenges in sliding mode controller therefore reducing the chattering is a major objective in this research. Chattering phenomenon is caused to the hitting in driver and mechanical parts therefore reduce the chattering is very important in this research. To reduce the chattering this research is focused on saturation function instead of

switching function. Figure 17 shows the power of boundary layer (saturation) method to reduce the chattering in sliding mode controller.

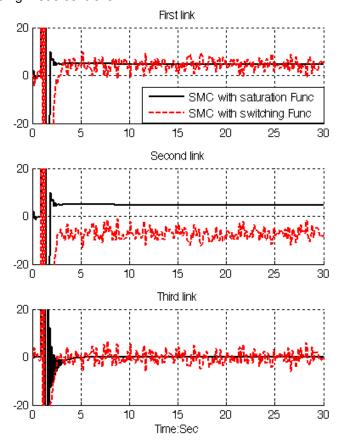


FIGURE 17: SMC with boundary layer Vs SMC with switching function: chattering phenomenon

Torque Performance

Figure 18 and 19 have indicated the power of chattering rejection in SMC with boundary layer, CTC and SMC with switching function with 40% disturbance and without disturbance.

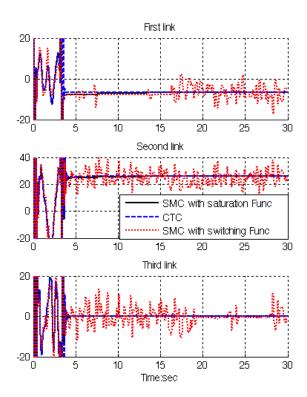


FIGURE 18: SMC with boundary layer, SMC with switching function and CTC for first, second and third joints torque performance without disturbance

Figure 18 shows torque performance for first three links PUMA robot manipulator in CTC, SMC with switching function and SMC with boundary layer without disturbance. Based on Figure 18, The SMC with boundary layer and CTC give considerable torque performance when compared to SMC with switching function. Figure 19 have indicated the robustness in torque performance for first three links PUMA robot manipulator in CTC, SMC with switching function and SMC with boundary layer in presence of 40% disturbance. In this research boundary layer sliding mode controller has the steady torque oscillation in presence of external disturbance compared to CTC and SMC with switching function therefore SMC with boundary layer is more robust then the other two controllers.

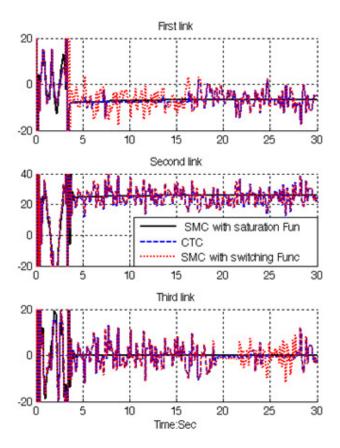


FIGURE 19: SMC with boundary layer, SMC with switching function and CTC for first, second and third link joints performance with40% disturbance

Based on Figure 19, it is observed that however all of three controllers have oscillation but SMC with boundary layer has better performance compared to CTC and SMC with switching function in presence of 40% disturbance. This is mainly because SMC with saturation function is more robust than CTC and SMC with switching function. Oscillation for all three controllers is increased but SMC with saturation function is more robust however is not still must be improved. In this part SMC with saturation function, CTC and SMC with switching function have been comparatively evaluation through MATLAB simulation, for PUMA robot manipulator control. It is observed that the SMC with saturation function gives the best performance in presence of uncertainties and external disturbance.

6. CONCLUSION

Refer to this research, position sliding mode controller (SMC) and computed torque controller (CTC) are proposed for PUMA robot manipulator. The first problem of the pure sliding mode controller with switching function was chattering phenomenon in certain and uncertain systems. The chattering phenomenon problem can be reduced in certain system by using linear saturation boundary layer function in sliding mode control law. The simulation results exhibit that the sliding mode controller with saturation function works well in certain system. This methodology is based on applied saturation function in sliding mode methodology to estimate chattering phenomenon. The results demonstrate that the sliding mode controller with saturation function is a model-based controllers which works well in certain and partly uncertain system. The stability and convergence of the sliding mode controller based on switching function is guarantee and proved by the Lyapunov method. Based on theoretical and simulation results, it is observed that sliding mode controller is more robust than computed torque controller for robot manipulator.

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